

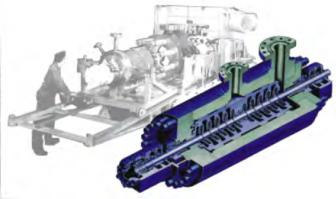
API 610 PUMP REPAIRS AND UPGRADES

Presented by Tom Newman

Manager, Technical Services and Business Development, Aftermarket Engineering

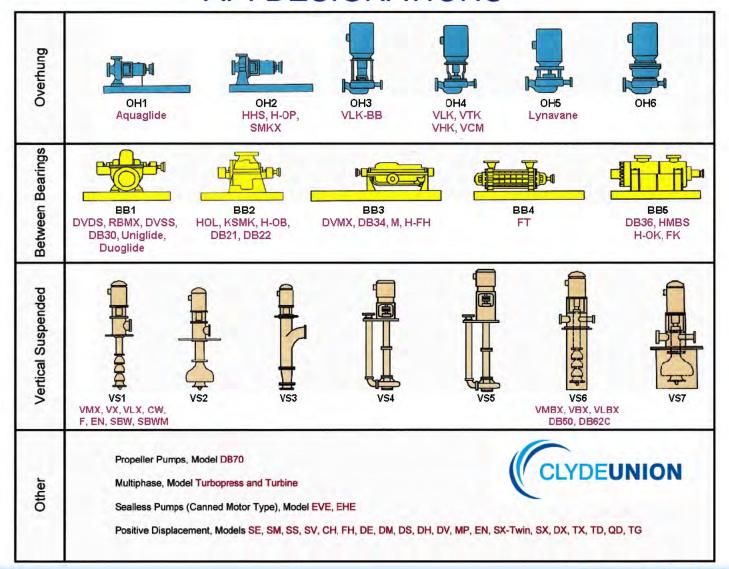








API DESIGNATIONS

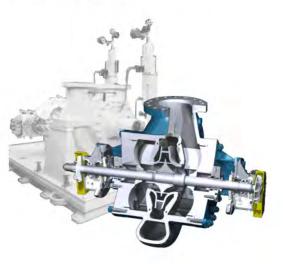


Repair/Upgrade Opportunities



- General Repair/Documentation
- Piping Upgrades/Gusseting
- Hydraulic Re-rates
- Material Upgrades
- Drop-In Replacement Pumps
- Back Pull-Out
- •360 Degree Bearing Housing Supports
- Emission Control
- OH4 Upgrades
- Field Testing
- A&Q.







General Repair / Documentation



Major Pipeline



Equipment Information

Pump Make: ABC Pump Pump Type: 3 x 6 x 9 BB-3 Serial Number: S/N999999 Equipment Number: P-101

Repair Job Reference: ABC123

KEY ITEMS TO INCLUDE

- Table of Contents
- Scope of Work
- Quality Documents
 - Inspection Test Plan
 - Weld Procedures
 - · Heat Treat Procedures
 - Material Specifications
- Inspection Reports
 - · As Found Reports
 - Testing/NDE
 - Balance Report
 - · As Built Report
 - · Heat Treat Records
 - Material Test Reports
- Updated Bill of Material
- Applicable Drawings
- In Progress Photos

General Repair / Documentation - ITP





Pump Model: 18x18x27 DB-30

Inspection & Test Plan

Page 2 of 3

TASK DESCRIPTION	PROCEDURE	PROCESORE CRITERIA		DOCUMENT APPLICABLE TO		NOTES	
Pump							
Weld Repair Casing Crack and Wear ring Fit Area	S-209	ASME	Report	Lower Pump Casing	U.H.C		
Post Weld Heat Treat	PWHT S-209	ASME	Report	Lower Pump Casing	U, C		
Liquid Penetrant Inspection (LPI)	GES 3008.00	ASME	Certificate	Lower Pump Casing Repairs	U, C		
Ultrasonic Inspect Casing Weld Repair (UT)	CES 3016.00	ASME	Certificate	Lower Pump Casing Repairs	U.C		
Radiographic Inspection of Weld Repairs if LPI / UT is inconclusive (RT)	CES 3015.00	ASME	Certificate	Lower Pump Casing Repairs	U, C		
Review Weld Inspection documents before Hydrostatic Test	Per Shell DEP requirements	API 610 & Shell DEP	Certificate	Lower Pump Casing Repairs	H, W		
Casing Machining	Per Drawings	Drawing	CU Internal Work	Casing halves	•		

Order

Certificate

Perf. Curve

Data / Report

Report

Perf. Curve

Certificate

Certificate

Certificate

As Built Report

Item No: 480-PU-003 A

ACCEPTANCE

Tolerances

API-810 85 ed.

API-810 8" ed.

API-610 8" ed.

N/A

API-810 8" ed.

ASTM E112

API-610

ASTM Standard

N/A

LEGEND:

Hydrostatic Test

Performance Test

Vibration Spectra

Hardness Check - Shaft

Material Certification

As Found Data (TDIQ)

Hardness Check - Wear Rings

NPSH Test

H - Hold "By UP" - Internal QS requirements only.

"By customer" - Manufacturing will not proceed without customer

CES 3001.00

CES 3002.01

CES 3002.01

N/A

CES 3002.01

Vendor Standard

Vendor Standard

Vendor Standard

W - Witnessed

Hydrodynamic bearing check after test

Manufacturing will not proceed until customer has witnessed the

C - Certified Certificate or Report will be provided as part of final data package.

BUR-24c Rev 4 Feb-04-05

U - Unwitnessed

No customer witness requirements identified. Certificate or Report will be provided as

· W.C

U,H,C

U.H.C

U. H. C

U,H,C

U.C

U.C

· U.C

part of final data package.

Casing, cover

Pump Performance

Pump Performance

Pump Performance

Shaft

Wear Rings

Impeller, Shaft

Mechanical Performance

Pump Serial No: CA4626C 104

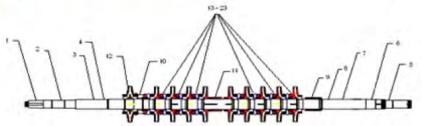
Activity carried out to satisfy UP internal quality system requirements.

General Repair / Documentation – As Found





As Found Inspection Report **Rotor TIR**



1. TIR @ Coupling 2. TIR @ Radial Brg. 3. TIR @ IB Seal Box 4. TIR @ IB Throat Bushing

 TIR @ OB THI GET BUSHING
 TIR @ Comp. Bushing
 TIR @ 1st Stg Pc
 TIR @ Center Bushing
 TIR @ Impeller 5. TIR @ Thrust Collar 6. TIR @ Thrust Brg.

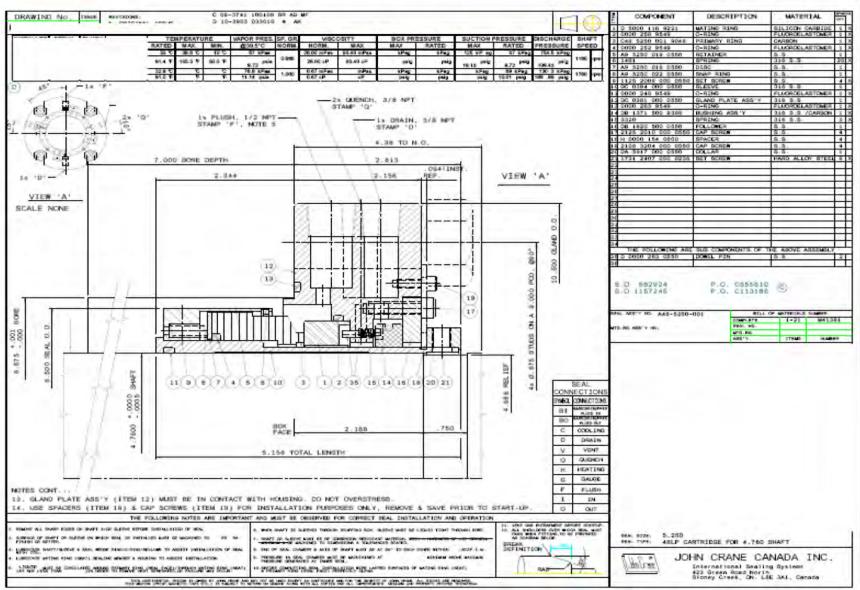
7. TIR @ OB Seal Box 8. TIR @ OB Throat Bushing

ROTOR RUN-OUT	Status		Status	
1			Impeller Stage	
2		12	Suction	
3		13	2	
4		14	3	
5		15	4	
6		16	5	
7		17	6	
8		18	7	
9		19	8	
10		20	9	
11		21	10	
		22	11	
		23	12	

	Status
R-	Rework
S-	Scrap
OK-	Okay

General Repair / Documentation - Seal Drawing





General Repair / Documentation – As Built

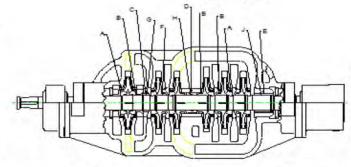


1	NOTUNION
11 0	CLYDEUNION
	and and a second a

As Built Inspection Report Rotor

Date.	
Customer:	
Job Number:	
Serial Number:	
Pump Type:	
Pump Size:	

No. of Stages: ______
Inspected By:



Description
A-Case View Ring (Dto Impelier Wear Ring Of Front)
Front
Grove Wear Ring (Dto Impelier Wear Ring Of

(Bed)
O to Stage Pece I Dto for Stage ServeCO
D-Center Bushing I Dto Center Seale CO
E-Compensating Bushing ID to Compensating

-impelier Bare Ditto Impelier Journal CD Rt.
-impelier Seque Ditto Bhaft CD Rt.
+Centre Seque Ditto Bhaft CD Rt.
-Compensating Seque ID to Bhaft CD Rt.

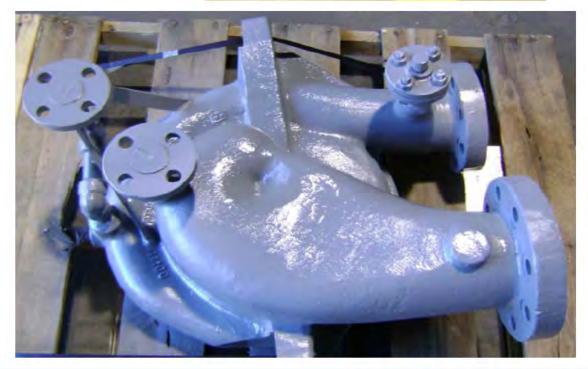
Stage	Starting Point	DESCRIPTION	SIZES (F)	SIZES (B)	DESCRIPTION	SIZE S
Suction		Casing Ring ID			ImpellerID	
Imp. Diameter		Impeller Ring OD			Shaft OD	
No. of Vanes		Clearance			Clearance	
		1st Stage Piece ID			1st Stage Sleeve ID	
1st Stage Piece		1st Stage Sleeve OD			Shaft OD	
A TAKE VV		Clearance			Clearance	
		Casing Ring ID			ImpellerID	
Imp. Diameter		Impeller Ring OD			Shaft OD	
No. of Vanes						
		Casing Ring ID			ImpellerID	
Imp. Diameter		Impeller Ring OD			Shaft OD	
No. of Vanes						
2nd		Casing Ring ID			ImpellerID	
Imp. Diameter		Impeller Ring OD			Shaft OD	
No. of Vanes		Clearance			Clearance	
3rd		Casing Ring ID		S	ImpellerID	
Imp. Diameter		Impeller Ring OD			Shaft OD	
No. of Vanes		Clearance			Clearance	

Piping Modifications



- Upgrade NPT connections to socket weld
- Two plane gusseting of piping
- Applicable to WCB, CA6NM and 300 SS
- Reduce risk of fires on older high temperature pumps



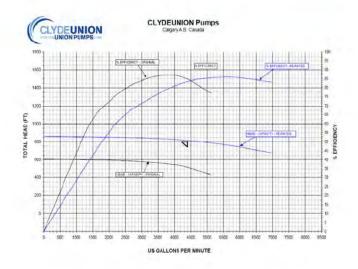




Hydraulic Re-Rate Classifications:

- Impeller Modifications:
 - Trimming
 - De-Staging
 - Under File (U/F)
 - Inlet Vane Cutback
- Casing/Volute Modifications:
 - Volute Lip Cutback
 - Angled Volute Lip Cutback
 - Volute Lip Extension
 - Custom Hydraulics





HYDRAULIC RE-RATES



AFFINITY TRIM LAWS

$$FLOW = \frac{Q1}{Q2} = \frac{D1}{D2}$$

$$HEAD = \frac{H1}{H2} = \left[\frac{D1}{D2}\right]^2$$

BHP =
$$\frac{BHP1}{BHP2} = \left[\frac{D1}{D2}\right]^3$$

CLYDEUNION TRIM LAWS

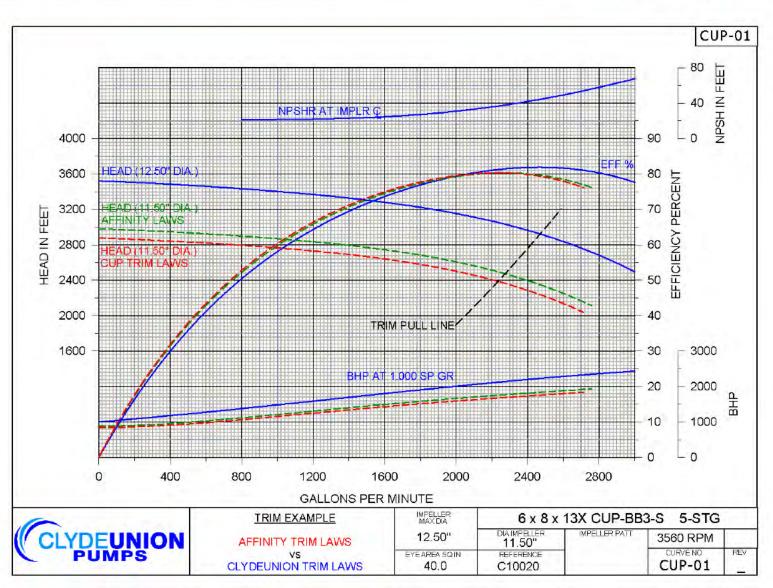
FLOW =
$$\frac{Q1}{Q2} = \frac{D1}{D2}$$
 +/- Variable

HEAD =
$$\frac{H1}{H2} = \left(\frac{D1}{D2}\right)^2$$
 +/- Variable

BHP =
$$\frac{BHP1}{BHP2} = \left(\frac{D1}{D2}\right)^3$$
 +/- Variable

HYDRAULIC RE-RATES

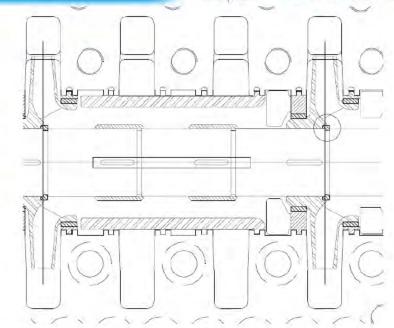


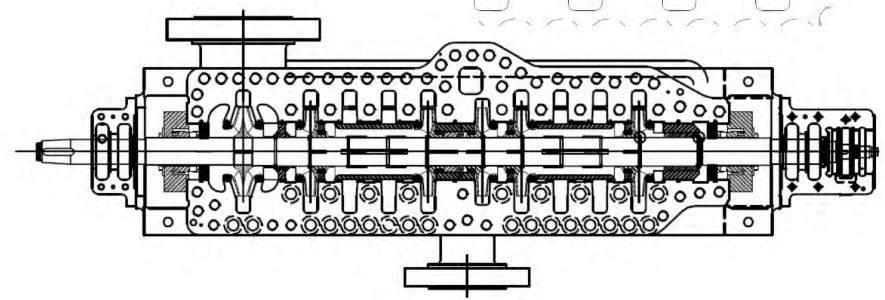


HYDRAULIC RE-RATES, Pump De-staging



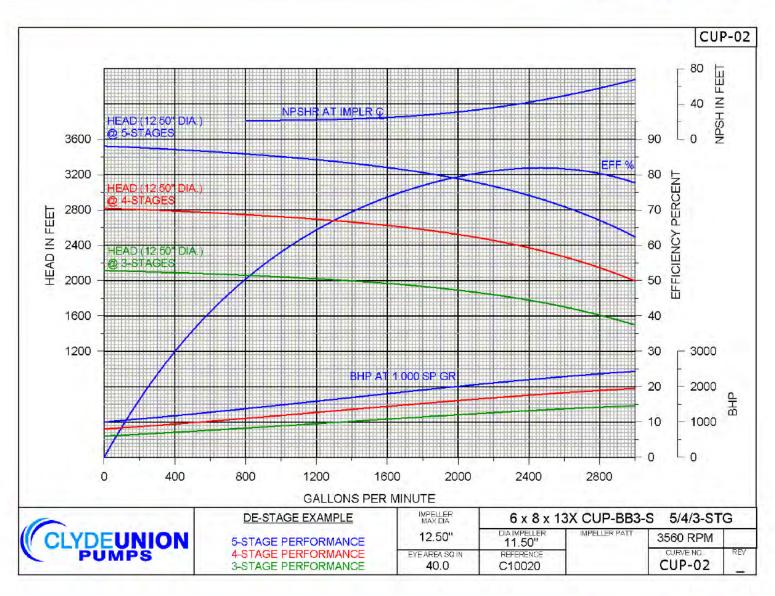
It is not unusual for pipelines to start up at low pressure then later change to high pressure at constant throughput. This change in pump head requirements can be suitably handled by selecting a multi-stage pump for the ultimate high pressure condition. For the initial low pressure condition, appropriate impellers are removed and the inter-stage chambers isolated by de-staging cylinders and sleeves. When changing to high pressure the de-staging cylinders and sleeves are removed and additional impellers installed (see illustrations below)





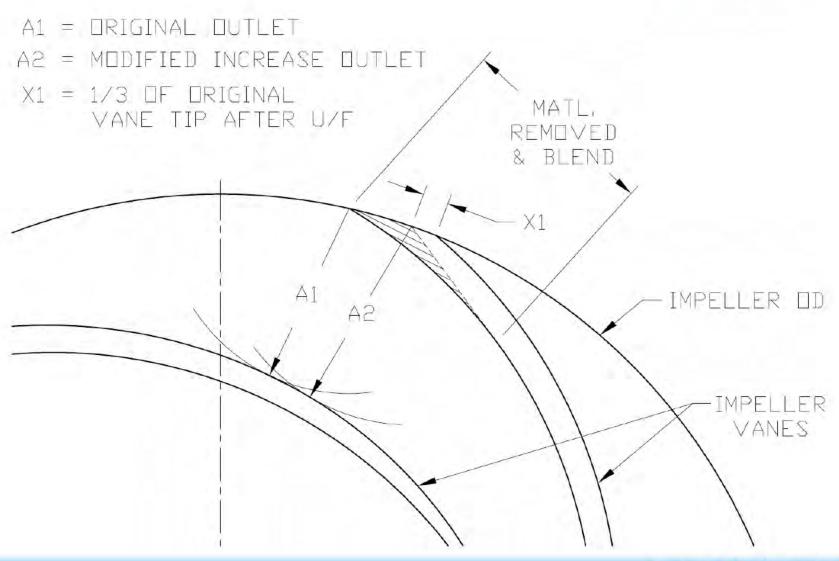
HYDRAULIC RE-RATES, Pump De-staging





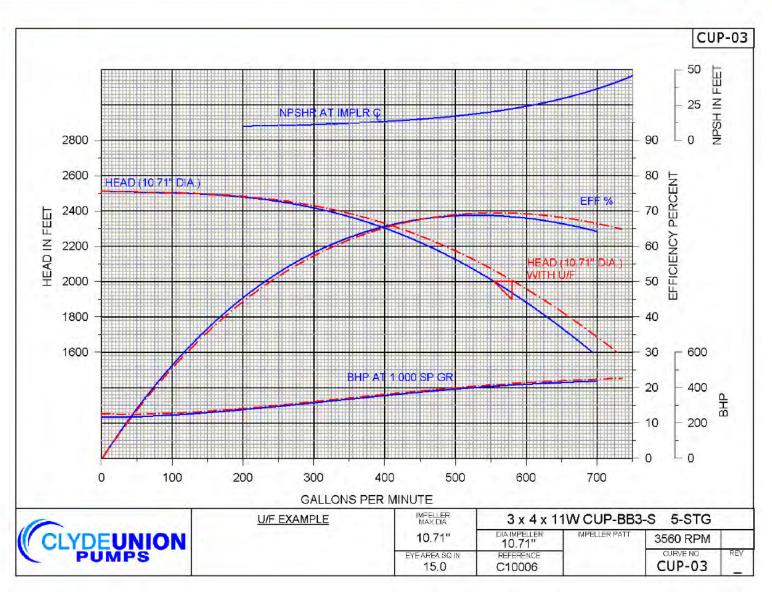
Hydraulic Re-Rates, Impeller Underfile (U/F)





Hydraulic Re-Rates, Impeller Underfile (U/F)





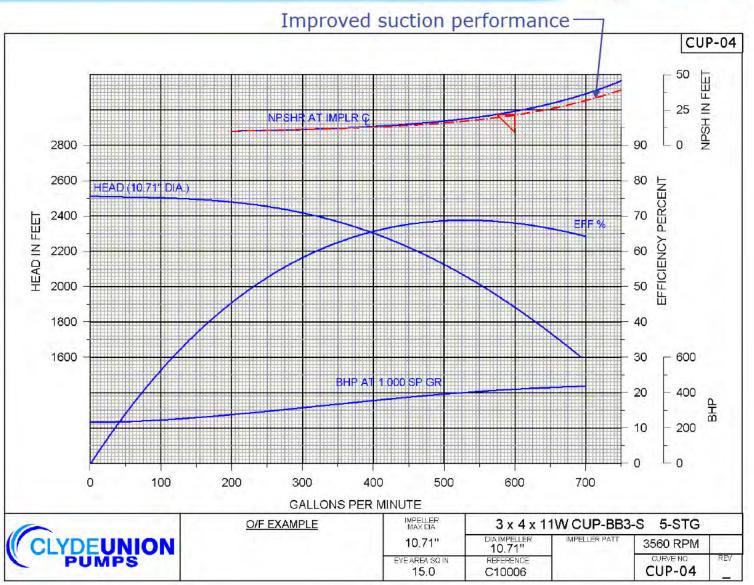


Cutback and file from the back (non-visible) side of the suction vane inlet

Cutback & File of Suction Vanes (1) Open passages between U/F suction (2) Thin the vanes at entry vane to given dimension and blend (a) Impeller Suction Vane (1) Cut-back leading edge (2) Thin the vanes at entry Cut back to given Filed to Knife edge dimension (b) Before modifications NPSH % Modified 100 100 Q %

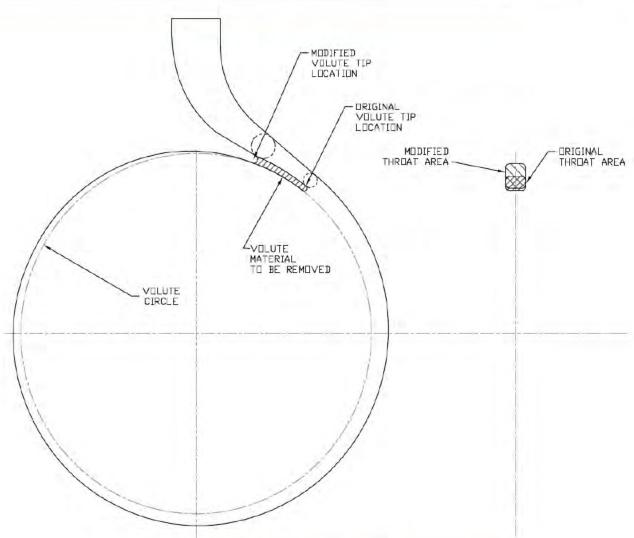
Hydraulic Re-Rate, Inlet Vane Cutback





Hydraulic Re-Rate, Volute Lip Cutback

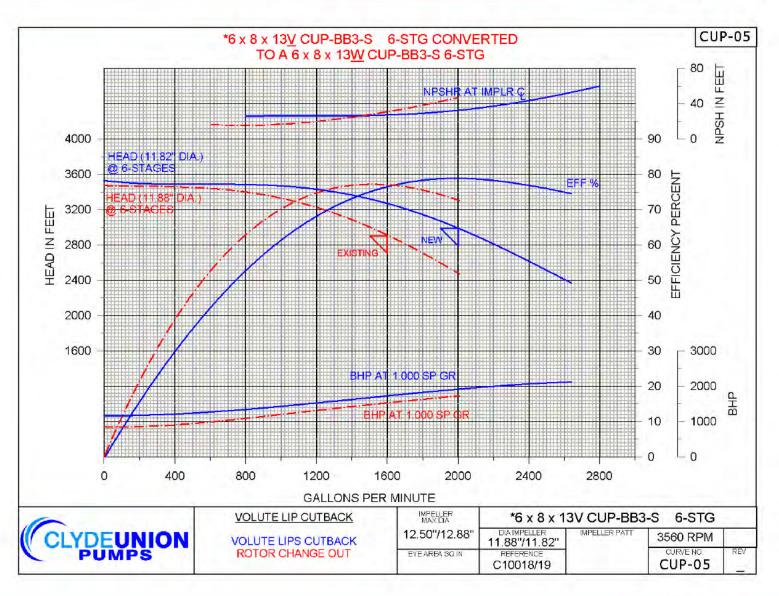




TYPICAL VOLUTE LIP CUTBACK

Hydraulic Re-Rate, Volute Lip Cutback







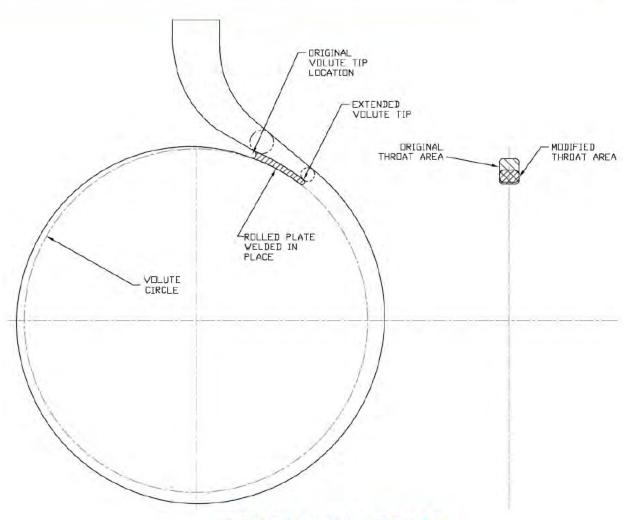
VOLUTE 'V' NOTCHING



Restore volute and cutback

Hydraulic Re-Rate, Volute Lip Extension

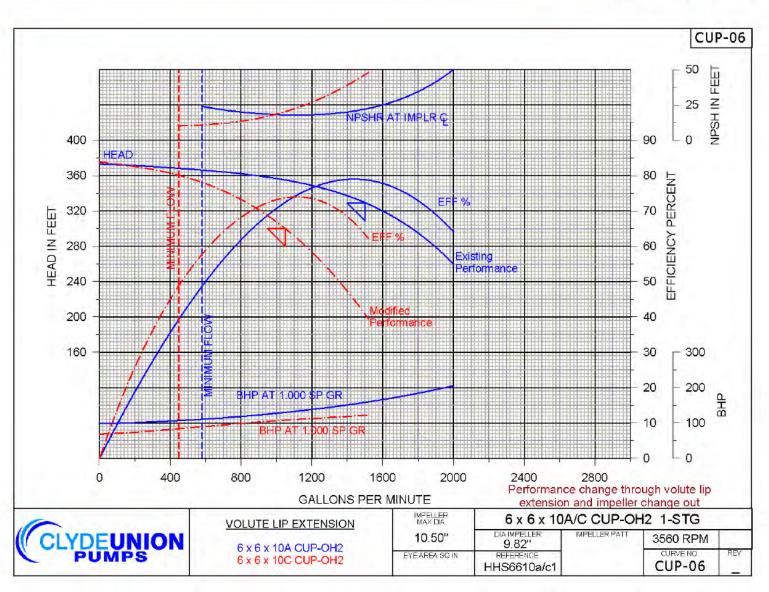




TYPICAL VOLUTE LIP EXTENSION

Hydraulic Re-Rate, Volute Lip Extension





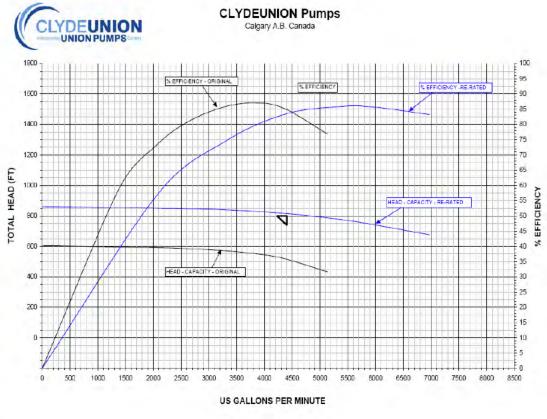
Hydraulic Re-Rate, Custom Hydraulics



Re-rate a United S-10 X 14 DVS (BB1) to meet new system curves based on four products with varying viscosities. The motor horsepower would be increased from 800 to 1250, with a goal of maximizing flow based on new available horsepower, system curves, and viscosity range from 45 cSt to 981 cSt

- Design and Manufacture a new impeller
- Hydraulic modifications to the existing volute
- Base plate modifications to accept new motor





MATERIAL UPGRADES



- Change API Material Designation
- Non-Metallic Wear Parts
 - PEEK
 - Vespel
 - Graphalloy
- Cladding/Weld Overlay
 - Inconel Overlay
 - Monel Overlay
- Wear Resistant Coatings



Material Upgrades



Table H.1 — Material classes for pump parts

PART			Material classes and abbreviations													
		1-1	1-2	S-1	S-3	S-4	S-5	S-6	S-81	S-91	C-6	A-7	A-8	D-1 i	D-2 J	
	Full compli- ance	Cla	CI	STL	STL	STL	STL	STL	STL	STL	12 % CHR	AUS	316 AUS	Duplex	Super Duplex	
	materials	CI	BRZ	CI	NI- RESIST		STL 12 % CHR	12 % CHR	316 AUS	Ni-Cu Alloy	12 % CHR	AUS C, d	316 AUS ^d	d Duplex	Supe Duple:	
Pressure casing	Yes	Cast iron	Cast iron	Carbon steel	Carbon steel	Carbon steel	Carbon Steel	Carbon Steel	Carbon Steel	Carbon Steel	12 % CHR	AUS	316 AUS	Duplex	Super Duplex	
Inner case parts: (bowls, diffusers, diaphragms)	No	Cast iron	Bronze	Cast fron	Ni-Resist	Cast iron	Carbon Steel	12 % CHR	316 AUS	Ni-Cu Alloy	12 % CHR	AUS	316 AUS	Duplex	Super Duplex	
impeller	Yes	Cast iron	Bronze	Cast (rón	Ni-Resist	Carbon steel	Carbon Steel	12 % CHR	316 AUS	Ni-Cu Alloy	12 % CHR	AUS	316 AUS	Duplex	Super Duplex	
Case wear rings ^K	Nio	Cast iron	Bronze	Cast	Ni-Resist	Cast	12 % CHR Hardened	12 % CHR Hardened	Hard-faced 316AUS ^e	Ni-Cu Alloy	12 % CHR Hardened	Hard-faced AUS ^e	Hard-faced 316 AUS ^e	Hard-faced Duplex ⁶	Hard-faced Super Duplex ⁹	
Impeller wear rings ^k	No	Cast iron	Bronze	Cast Iron	Ni-Resist	Cast	12 % CHR Hardened	12 % CHR Hardened	Hard-faced 316ÅUS ⁸	Ni-Cu Alloy	12 % CHR Hardened	Hard-faced AUS ⁸	Hard-faced 316 AUS ^e	Hard-faced Duplex e	Hard-faced Super Duplex ^e	
Shaft d	Yes	Carbon steel	Carbon steel	Carbon steel	Carbon steel	Carbon steel	AlSI 4140	AISI 4140 f	316 AUS	Ni-Cu Alloy	12 % CHR	AUS	316 AUS	Duplex	Super Duplex	
Throat bushings k	No	Cast iron	Bronze	Cast iron	Ni-Resist	Cast	12 % CHR Hardened	12 % CHR Hardened	316 AUS	NI-Cu Alloy	12 % CHR Hardened	AUS	316 AUS	Duplex	Super Duplex	
Interstage sleeves ^k	No	Cast iron	Bronze	Cast	Ni-Resist	Cast	12 % CHR Hardened	12 % GHR Hardened	Hard-faced 316AUS e	Ni-Cu Alloy	12 % CHR Hardened	Hard-faced AUS ⁹	Hard-faced 316 AUS ^e	Hard-faced Duplex ^e	Hard-faced Super Duplex ^è	
Interstage bushings k	No	Cast from	Bronze	Cast	Ni-Resist	Cast	12 % CHR Hardened	12 % CHR Hardened	Hard-faced 316AUS ⁸	Ni-Cu Alloy	12 % CHR Hardened	Hard-faced AUS ⁹	Hard-faced 316 AUS ⁹	Hard-faced Duplex ^e	Hard-faced Super Duplex ⁶	
Case and gland studs	Yes	Carbon steel	Carbon	AISI 4140 steel	AISI 4140 Steel	AISI 4140 Steel	AISI 4140 Steel	AISI 4140 Steel	AISI 4140 Steel	Ni-Cu Alloy Hardened i	AISI 4140 Steel	AISI 4140 Steel	AISI 4140 Steel	Duplex i	Super Duplex	
Case gasket	No	AUS. Spiral wound ⁹	AUS, Spiral wound ⁹	AUS. Spiral wound ^g	AUS. Spiral wound ⁹	AUS, Spiral wound ⁹	AUS, Spiral wound ^g	AUS, Spiral wound ⁹	316 AUS Spiral wound ⁹	Ni-Cu Alloy, Spiral wound, PTFE filled ⁹	AUS, Spiral wound ⁹	AUS, Spiral wound ^g	316 AUS Spiral wound ⁹	Duplex SS Spiral wound ^g	Duplex SS Spiral Wound ^g	

Material Upgrades, Non-Metallic Wear Parts





ADVANTAGES:

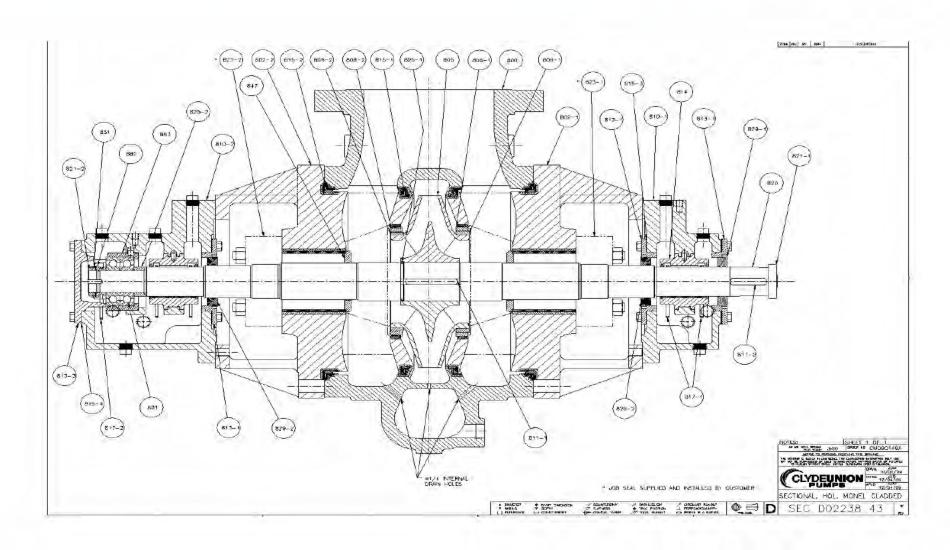
- Non-Galling Materials
- Increased efficiency with reduced clearances
- Chemical Resistance

DISADVANTAGES:

- Cost
- Temperature Limitations
- Installation Difficulties

Material Upgrades, Cladding/Overlay





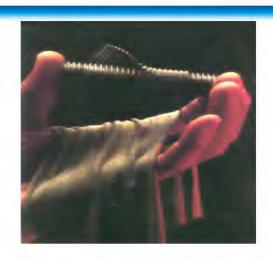
Material Upgrades, Wear Resistant Coatings



- Tungsten Carbide (WC) Spray, HVOF
 - Line of site process
 - Low bond strength (10,000 12,000 psi) easily undermined
- Boron Diffusion (BD)
 - Only 0.008-0.010" thickness on carbon steel
 - Even thinner ~0.003" on 12-chrome
 - Large abrasive particles can puncture the coating
 - Once thin coating punctured, the substrate is left without wear protection
- Weld Overlay
 - Soft binder holding carbide particles
 - Uneven carbide distribution in the matrix Preferential erosion path
- Solid Stellite impellers
 - Difficult to source consistent castings
- Welded Stellite
 - Delayed cracking potential
 - Low carbide content (12%-15%) Not optimum protection for erosion wear
- Infiltration Brazed Tungsten Carbide Cladding Technology
 - Flexible, confirms to complex geometries
 - Not limited to line of site application
 - Provides premium wear protection in previously difficult to reach locations

Material Upgrades, Infiltration Brazed Tungsten Carbide Cladding



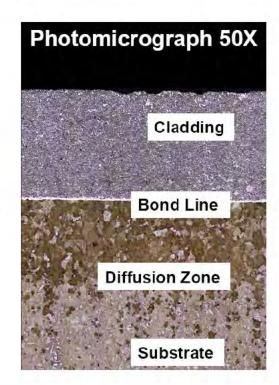


The Process

- Flexible Tungsten Carbide cloth can conform to complex geometries.
- The cloth is adhered to the targeted wear surface on the substrate and over-layered with a brazen cloth of filler metal.
- Then placed in a vacuum furnace until the brazing material melts and infiltrates the tungsten carbide cloth.

Lessons Learned

- Preferred base material is carbon steel versus CA6NM since any welding done to CA6NM after the coating process would require heat treatment.
- Pre-heating of large castings using temperatures similar to the coating process has been found to reduce the warpage of the part during the coating process.
- Balancing of the impeller after the coating process requires metal to be added by using tungsten carbide weld rod. It has been found that large impellers can require a large amount of weld material to be added to obtain final balance requirements. The large amount of weld has caused axial clearance issues during final assembly and has also affected the pump performance.
- The affect of the coating on pump performance needs further review and actual testing. To date the coating has only been used and performance tested on one size of a HOL pump. The affect on performance on smaller pumps is unknown.







Original Material 6 Weeks



Infiltration Brazed Tungsten Carbide Cladding
12 Weeks

Clarified oil with heavy catalyst slurry

- Prior to cladding the unit was scrapped after one year of service
- After one year with clad technology, 0.12 0.2 mm Material Loss
- Pump Life Extended from 1 Year to 3-4 Years

Drop-In Replacement Pumps



Design and Manufacture replacement pump

- Match existing mounting dimensions and nozzle locations
- Provide custom or standard hydraulics
- Provide mechanical upgrades including:
 - Bearing Housing Upgrades
 - Mechanical Seal Upgrades
 - Material Upgrades



Drop-In Replacement Pump with IBTCC Coating

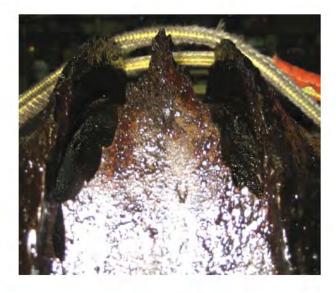


Gulf Coast Refinery – Original FCCU Catalyst Slurry Pump





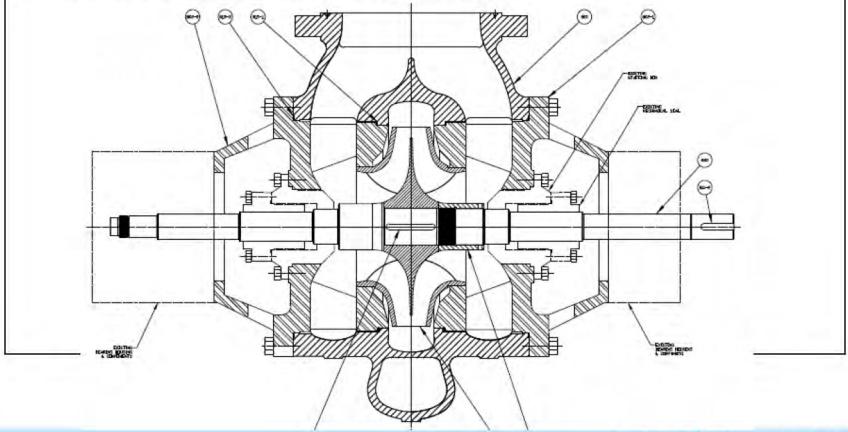




Drop-In Replacement Pump with IBTCC Coating



- Drop-In Design No piping or foundation modifications
- Wear Plates, High Chrome Iron Impeller, and stuffing box inserts eliminated
- All wetted parts are IBTCC coated
- 360 degree bearing support upgrade included
- New hydraulics to match current operating conditions



Drop-In Replacement Pump with IBTCC Coating



Gulf Coast Refinery - Replacement FCCU Catalyst Slurry Pump







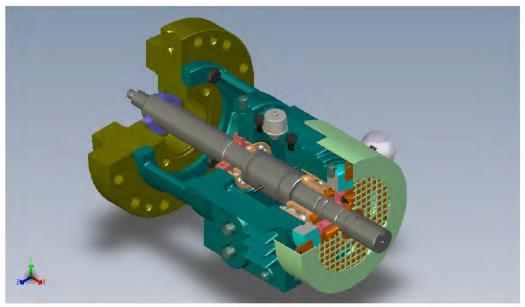


OH2 Back Pull Out



Typical API-610 10th Edition OH2 Back Pull Out (BPO)





OH2 Back Pull Out



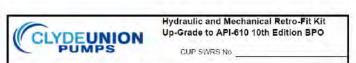
Back Pull Out (BPO) Advantages

Features	Benefits	
Direct drop-in replacement	Minimum installation cost	
Seal chamber meets API 682 Standards	Extended seal life & reduced emissions.	
Large shaft diameter with lower L3/D4 ratio	Improved mechanical seal life	
Designed for outside drive collar and cartridge mech. seals	Ease of seal setting and maintenance	
Oiling pin lubrication	Reduced oil contamination and extended bearing life	
7300 series angular contact thrust bearings	Extended bearing Life	
Finned bearing housing	Improved cooling and extended bearing life	
External fan for cooling housing	Eliminates cooling water to 700 F	

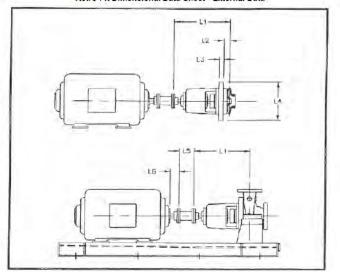
OH2 Back Pull Out



Hydraulic & Mechanical Retro-fit Kit Site Inspection Forms



Retro-Fit Dimensional Data Sheet - External Data



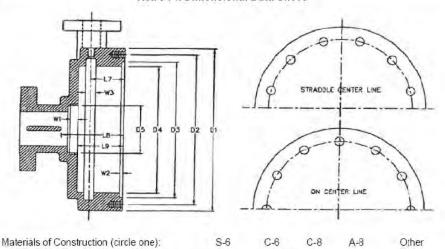
Dimension Description	Dimension	Measurement (in)	Notes
Case CL to Pump Shaft End	Lt		
Cover Flange Face to CL of Imp.	Ĺ2		
Cover Flange Thickness	L3		
Cover Flange OD	1.4		
Coupling Spacer Length	L5		
Motor Shaft Extension	L6		



Hydraulic and Mechanical Retro-Fit Kit Up-Grade to API-610 10th Edition BPO

CUP SWRS No:

Retro-Fit Dimensional Data Sheet



Dimension Description	Dimension	Measurement (inches)	
Casing OD	D1		
Bolt Circle Diameter	D2		Bolt Circle (check one) Straddles CL On CL
Gasket Fit ID	D3		
Case ID	D4		
Wear Ring ID	D5		
Case Face to CL of Volute	L7		
Case Face to Flow Splitter	L8		
Case Face to Wear Ring	L9		
Case Wear Ring Width	W1		
Gasket Fit Depth	W2		
Volute Width	W3		

360 Degree Bearing Housing Supports







- 360 Degree Bearing Housing Support/Bearing Housing Upgrades
 - Reduced Vibration
 - Increased Stability
 - Decreased Deflection during warm-up
 - Bearing Options Ball/Ball, Sleeve/Ball, Sleeve/Kingsbury
 - Lubrication Options Oil Mist, Ring Oil, Pressure Lube
 - Cooling Options Fan Cooled, Water Cooled

Emission Control



CP System

Constant Pressure Barrier Fluid Sealing System

Improve Safety, Reliability and Eliminate Fugitive Emissions

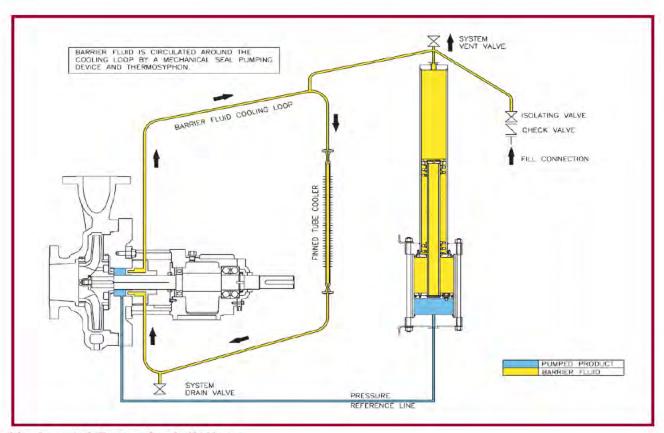
- Zero leakage to atmosphere.
- Requires no external power or nitrogen gas source.
- Maintains constant differential pressure across seals regardless of fluctuations in suction or discharge pressure.



Emission Control



CP SYSTEM

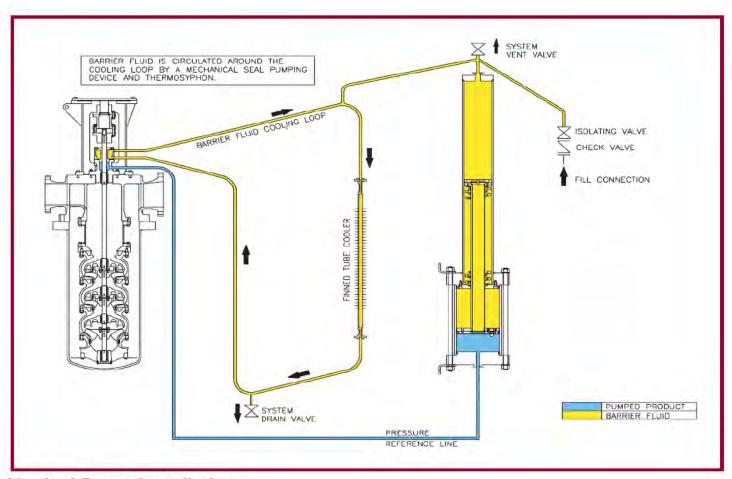


Horizontal Pump Installation

Emission Control



CP SYSTEM



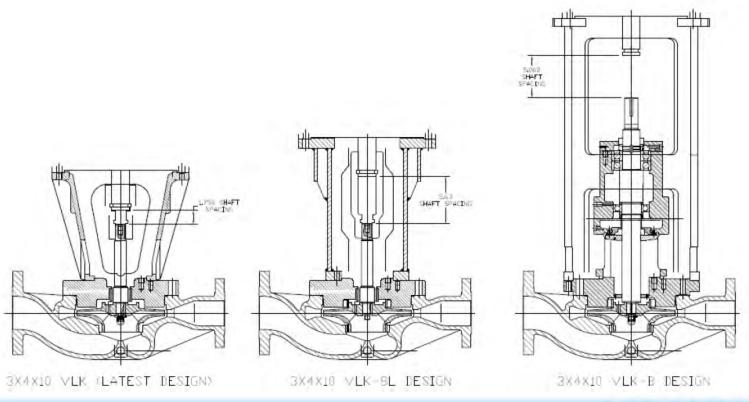
Vertical Pump Installation

OH-4 Upgrades



Common Mechanical Upgrades:

- •OH4 to OH4 with Extended Coupling: Ability to remove cartridge mechanical seal without removing the motor stand
- •OH4 to OH3: Eliminates rigid coupling in favor of bearing bracket design





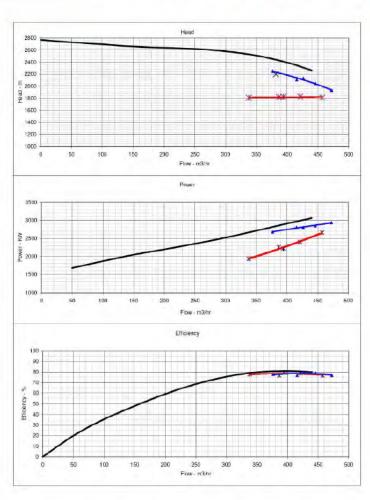
Why field test a pump?

- Field tests are run for the purpose of demonstrating satisfactory mechanical and hydraulic operation under actual conditions.
- Pumps do not operate in isolation. Rather, it is a part of a system and as such certain issues within the system can be affecting pump performance not seen on the test bed.
- Many field performance tests can be conducted without interruption to the pump or process operations.
- If the pump has been tested in the OE test lab before shipping, the field test Is used as verification that the equipment was repaired / re-rated properly.
- Field tests are used as a baseline for troubleshooting, energy efficiency studies, and life cycle cost analysis.



Basic Field Test Reports Includes:

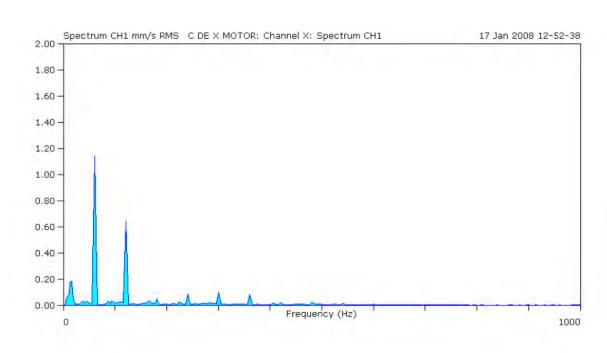
- Comparison of hydraulic and mechanical performance against known performance
- Assessment of any likely wear or known deterioration
- Recommendations for changes in operation or maintenance
- Recommendations for possible installation improvements
- Proposal for any pump modifications (where applicable) including mechanical upgrades or hydraulic re-rates





In depth reports include:

- Full FFT Vibration spectra analysis
- Energy efficiency assessment and possible cost savings
- System analysis
- Modal & resonance analysis





Site Photos (where allowed) are included in all reports illustrating test setup and assessment of pump condition and environment







